AMENDMENTS TO THE SPECIFICATION

In the Specification

Please substitute the following amended paragraphs and/or sections (deleted matter is shown by strikethrough and added matter is shown by underlining):

On page 6, paragraph [0023]:

The stationary housing 10 has a receiving opening 10a that passes through the central portion thereof (FIG. [[1]] 2). Also as shown in FIG. 3, the stationary housing 10 has an inner circular recessed portion in which guide grooves 10b and 10c are formed so as to perpendicularly extend vertically and horizontally. The operating cam 14 is disposed in the guide grooves 10b. The pair of slide pawls 16 (16A and a6B) are disposed in the guide grooves 10c.

On page 8, paragraph [0030]:

When the respective construction elements of the reclining mechanism R are assembled, the operating cam 14 is positioned in the cam guide grooves 10b of the stationary housing 10 so as to slide right and left in FIG.3. Conversely, the slide pawls of 16 are positioned in the pawl guide grooves 10c of the stationary housing 10 so as to oppose the operating cam 14 and to radially slide without circumferentially moving. The engagement projections 14a and 14b of the operating cam 14 are respectively positioned in the engagement recesses 16c of the slide pawls 16. Also, the cam portions 14c and 14d of the operating cam 14 are positioned so as to contact or to be contactable with the contact portions 16b of the slide pawls 16.

On page 9, paragraph [0032]:

Operation of the reclining mechanism R will now be described.

FIG. 3 shows an unlocking or free condition of the reclining mechanism R, i.e., a condition in which the rotational housing 12 can rotate. In this condition, the slide pawls 16 are positioned such that the respective toothed portions 16a faces the non-toothed portions 12c of the rotational housing 12. Also, in this free condition, the operating shaft 22 is not applied with a rotational operating force so that the operating cam 14 is applied with the actuating force in a leftward direction in FIG. [[2]] 3 by the force of the spring 24. Thus, the cam portions 14c of the operating cam 14 contact the contact portions 16b of the slide pawl 16A that is positioned at the upper side in FIG. [[2]] 3, thereby radially outwardly forcing the slide pawl 16A.

On page 9, paragraph [0033]:

As a result, the slide pawl 16A is radially restrictively positioned while the toothed portion 16a contacts the non-toothed portion 12c of the rotational housing 12. Therefore, the operating cam 14 in this condition is applied with a reactive force from the slide pawl 16A so as to be retained by wall surfaces A, positioned on one side (the lower side in FIG. 3) of the cam guide grooves 10b. In other words, the operating cam 14 radially outwardly forces the slide pawl 16A by utilizing the wall surfaces A of the cam guide grooves 10b as supporting points. Further, in this condition a narrow clearance is formed between wall surfaces positioned on the opposite side (the upper side in FIG. 3) of the cam guide grooves 10b and the operating cam 14. In addition, the cam portions 14d of the operating cam 14 are not contacting the contact portions 16b of the slide pawl 16B that is positioned at the lower side of FIG. [[2]] 3.

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On page 11, paragraph [0040]:

Next, the second embodiment will be described with reference to FIG. 8.

FIG. 8 is a plan view corresponding to FIG. 3 and illustrates a reclining mechanism according to the second embodiment. As will be apparent from this drawing, in the third second embodiment, the present invention is applied to a reclining mechanism R that includes a rotating-type operating cam and three slide pawls.

On page 12, paragraph [0042]:

FIG. 8 shows an unlocking or free condition of the reclining mechanism R, i.e., a condition in which the housings 10 and 12 can rotate relative to each other. In this condition, the cam portions 114c of the operating cam 114 contact the contact portions 116b of the slide pawl 116A, thereby radially outwardly forcing the slide pawl 116A. As a result, similar to the first embodiment, the slide pawl 116A is radially restrictively positioned while its toothed portion 116a contacts the non-toothed portion of the rachets 12b housing 12. Therefore, the operating cam 114 is applied with a reactive force from the slide pawl 116A so as to be retained by a portion of the inner circumferential surfaces of the bearing openings 13. In other words, the operating cam 114 radially outwardly forces the slide pawl 116A by utilizing the inner circumferential surfaces of the bearing openings 13 as supporting points. Further, the cam portions 114d and 114e of the operating cam 114 are not contacting the contact portions 116b of the slide pawls 116B and 116C.